

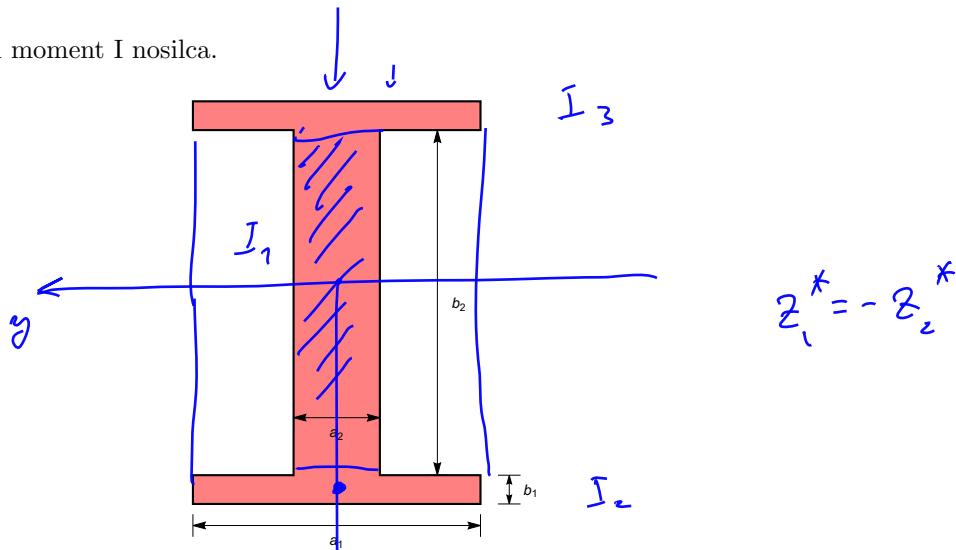
$$\underline{\sigma} = \frac{M}{I} z$$

$$J = \frac{S^2 A}{A}$$

Predavanje 19. maj 2021

$$\underline{J} = \frac{1}{12} a b^3$$

Primer: ploskovni moment I nosilca.



Slika 1: I nosilec.

$$J = J_1 + J_2 + J_3$$

$$J_1 = \frac{1}{12} a_2 b_2^3$$

$$J_2 = (z_2^*)^2 A_2 + \frac{1}{12} a_1 b_1^3 = \frac{1}{12} a_1 b_1^3 + \frac{1}{2} (k_2 + k_1)^2 a_1 b_1$$

$$J_3 = (z_1^*)^2 A_1 + \frac{1}{12} a_1 k_1^3 = J_2$$

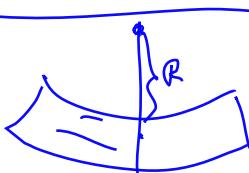
$$J = \frac{1}{12} a_2 b_2^3 + \frac{1}{12} a_1 b_1^3 + \frac{1}{2} (k_2 + k_1)^2 a_1 b_1 \leftarrow$$

Aproximacija za tankostenski nosilec

$$J = \frac{1}{12} a_2 b_2^3 + \frac{1}{2} a_1 b_1^2$$

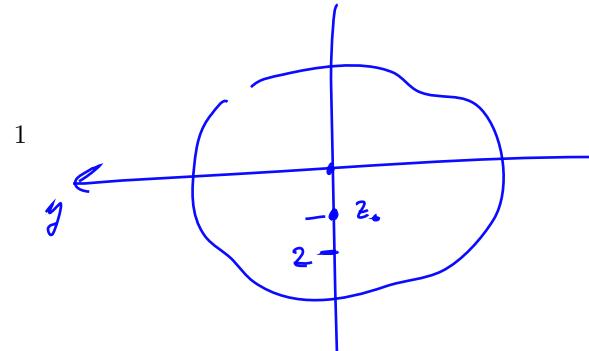
$$b_1 \ll b_2$$

$$\frac{M_2}{J} \ll \sigma_0$$



$$\epsilon = \frac{d}{R} = \frac{z - z_0}{R}$$

$$\sigma = E \epsilon = \frac{E}{R} (z - z_0)$$



$$0 = \int_A \sigma dA = \int_A \frac{E}{R} (z - z_0) dA = \frac{1}{R} \int_A E(z - z_0) dA \quad |z|$$

Določitev z_0 koordinate nevtralne osi v referenčnem položaju. Pogoj

$$\underbrace{0 = \int_A E(z - z_0) dA}_{\text{E konstanta}} \quad \text{II}$$

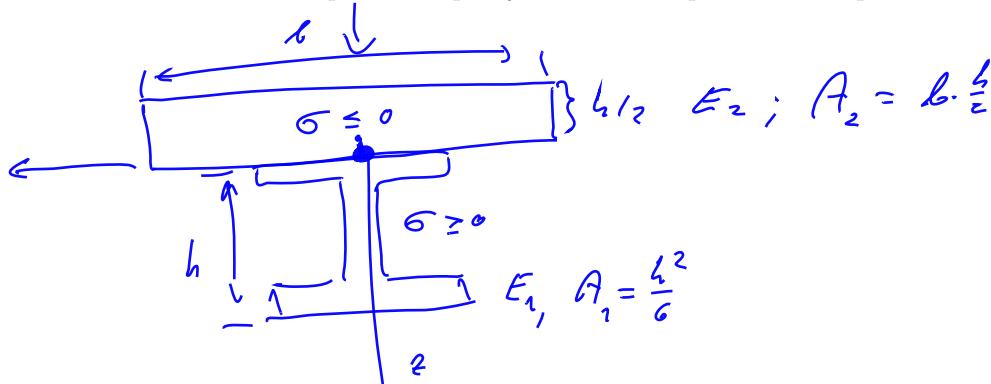
$$\int_A (z - z_0) dA = 0 ; \quad \boxed{\int_A z dA} = z_0 \int_A dA = z_0 |A|$$

$$\boxed{z_0 = z_x}$$

Za mehkostranski E (kompozitni nosilec) mora veljati

$$-\int_A E(z - z_0) dA = 0$$

Primer: na jekleni I nosilec višine h in površine $h^2/6$ je postavljena betonska plošča dimenzije $b \times h/2$. Določi b takoj, da bo v betonski plošči kompresijsko, v nosilcu pa natezno napetostno stanje.



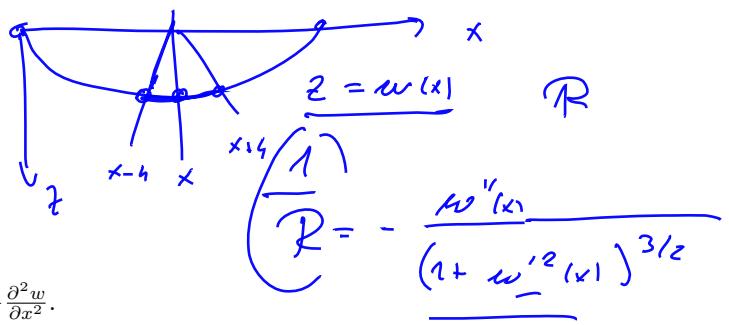
$$0 = \int_A E z dA = \int_{A_1} E_1 z dA + \int_{A_2} E_2 z dA = E_1 \int_{A_1} z dA + E_2 \int_{A_2} z dA$$

$$\boxed{z_1^* |A_1|} \quad \boxed{z_2^* |A_2|}$$

$$0 = E_1 \cdot \frac{h}{2} \cdot \frac{h^2}{6} + E_2 \left(-\frac{h}{4} \right) b \cdot \frac{h}{2}$$

$$E_2 \cdot \frac{1}{8} h^2 b = E_1 \frac{1}{12} h^3 \Rightarrow b = \frac{E_1}{E_2} h \cdot \frac{8}{12} = \frac{2 E_1 h}{3 E_2}$$





Upogib nevtralne osi

Upogib nevtralne osi $w(x)$, aproksimacija $\frac{1}{R} = -\frac{\partial^2 w}{\partial x^2}$.

$$\frac{1}{R} = -\frac{d^2 w}{dx^2}$$

$$\epsilon = \frac{z}{R}; \quad \sigma = E \frac{z}{R};$$

$$\frac{dM}{dx} = Q; \quad \frac{dQ}{dx} = -q(x)$$

$$\frac{d^2 M}{dx^2} = \frac{dQ}{dx} = -q(x)$$

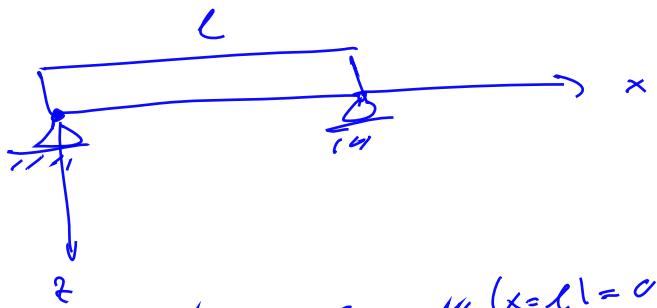
$$\frac{d^2 M}{dx^2} (EI) \frac{d^2 w}{dx^2} = q(x)$$

$$M = \frac{EI}{R} = -EI \frac{d^2 w}{dx^2}$$

$$\frac{d^2 M}{dx^2} = -q(x) = -\frac{d^2}{dx^2} (EI \frac{d^2 w}{dx^2})$$

$$EI \frac{d^4 w}{dx^4} = q(x)$$

Enačba upogiba $\frac{\partial^2}{\partial x^2} (EI \frac{\partial^2 w}{\partial x^2}) = q$.



Robni pogoji enačbe upogiba nosilca:

- členkasta podpora;
- konzolno vpetje;
- prosti konec;
- prosti konec;
- predpisana prečna obremenitev na koncu;
- predpisani upogibni moment na koncu.

$$x=l ; \quad M(x=l) = 0 \quad (\Rightarrow w''(x=l) = 0)$$

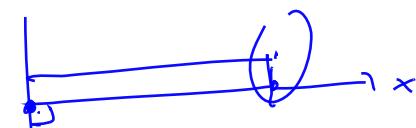
$$Q(x=l) = 0 \quad (\Rightarrow w'''(x=l) = 0)$$

$$\left(\frac{dM}{dx} = Q \right)$$

$$M = -EIw''$$

$$w(x=0) = 0 ; \quad w(x=l) = 0$$

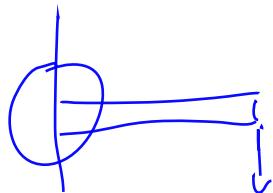
$$M(x=l) = 0 \quad (\Rightarrow w''(x=l) = 0)$$



$$w(x=0) = 0$$

$$w'(x=l) = 0$$

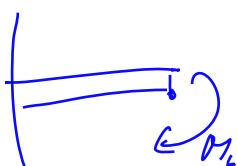
$$\begin{aligned} & \text{at } x=l \\ & w'(x=l) \\ & = -\tan \alpha \end{aligned}$$



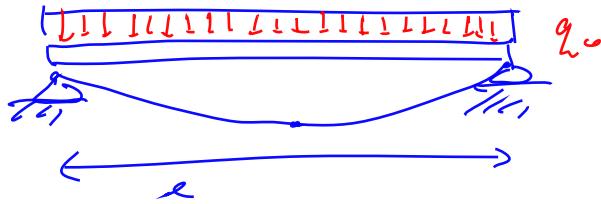
$$M = -EI \frac{d^2w}{dx^2} \quad \Rightarrow \quad -EI \frac{d^3w}{dx^3} = Q$$

$$w'''(x=l) = -\frac{Q}{EI}$$

$$w''(x=l) = 0$$



$$w'''(x=l) = 0 ; \quad w''(x=l) = -\frac{M_0}{EI}$$



Upogib enostavno podprtoga nosilca z linijsko obremenitvijo, določitev maksimalnega upogiba

$$w_{\max} = \frac{5q_0 l^4}{384 EI};$$

$$\underline{EI \frac{\partial^4 w}{dx^4} = q_0}$$

$$w''' = \frac{q_0}{EI} \Rightarrow w''' = \frac{q_0}{EI} x + C_1$$

$$\rightarrow w'' = \frac{1}{2} \frac{q_0}{EI} x^2 + C_1 x + C_2$$

$$w' = \frac{1}{6} \frac{q_0}{EI} x^3 + \frac{1}{2} C_1 x^2 + C_2 x +$$

$$\rightarrow w = \frac{1}{24} \frac{q_0}{EI} x^4 + \frac{1}{6} C_1 x^3 + \frac{1}{2} C_2 x^2$$

$$w = \frac{q_0}{24 EI} x^4 + D_1 x^3 + D_2 x^2 +$$

$$w(x=0) = 0 \Rightarrow C_1 = 0 \quad w''(x=0) = 0$$

$$w'(x=l) = 0 \Rightarrow \frac{q_0}{24 EI} l^4 + \frac{1}{6} = 0$$

$$w''(x=l) = 0 \Rightarrow \frac{q_0}{24 EI} l^2 + C_2 = 0$$

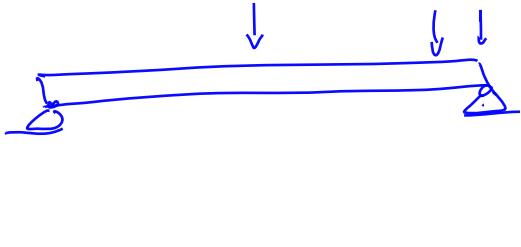
$$C_2 = -\frac{q_0 l^2}{24 EI} \quad C_3 = -\frac{q_0}{24 EI} l^3 + 6 \frac{1}{24 EI} l^3 = \\ = \frac{1}{12 EI} l^3 \left(1 - \frac{1}{2} \right) = \frac{q_0 l^3}{24 EI}$$

$$w = \frac{q_0}{EI} \left[\frac{1}{24} x^4 - \frac{1}{12} l x^3 + \frac{1}{24} l^3 x \right] = \\ = \left(\frac{q_0}{24 EI} l^4 \right) \left(\left(\frac{x}{l} \right)^4 - 2 \left(\frac{x}{l} \right)^3 + \frac{x}{l} \right)$$

$$w\left(\frac{l}{2}\right) = \frac{q_0 l^4}{24 EI} \left(\underbrace{\left(\frac{1}{2} \right)^4 - 2 \cdot \left(\frac{1}{2} \right)^3 + \frac{1}{2}}_5 \right)$$

$$= \frac{5q_0 l^4}{384 EI} = \frac{5 F l^3}{384 EI} \quad \frac{1}{2} \left(\frac{1}{8} - \frac{1}{2} + 1 \right) = \frac{1}{2} \frac{1 - 4 + 8}{8} = \frac{5}{16}$$

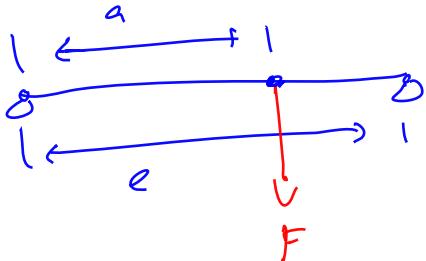
$$\frac{5}{384} \leq \frac{1}{58}$$



Upogib enostavno podprtga nosilca s točkovno obremenitvijo, določitev maksimalnega upogiba

$$w_{\max} = \frac{Fl^3}{48EI}$$

$$-EI \frac{\partial^2 w}{\partial x^2} = M$$



$$0 \leq x \leq a$$

$$a \leq x \leq 0$$

$$EI \underline{w'''}_x = 0$$

$$EI \underline{\frac{w''}{x}} = 0$$

$$w_c = C_1 x^3 + C_2 x^2 + C_3 x + C_4$$

$$w_d = D_1 x^3 + D_2 x^2 + D_3 x + D_4$$

$$w_c(a) = 0; \quad w_c''(a) = 0$$

$$w_d(l) = 0; \quad w_d'''(l) = 0$$

$$C_4 = 0; \quad C_2 = 0$$

$$w_c(a) = w_d(a)$$

$$w_c'(a) = w_d'(a)$$

$$w_c''(a) = w_d''(a)$$

$$w_c'''(a) = w_d'''(a)$$

$$w_c' = 3C_1 x^2 + 2C_2 x + C_3$$

$$w_d = D_1 \underline{(x-l)^3} + D_2 \underline{(x-l)^2} + D_3 \underline{(x-l)} + D_4$$

$$w_c'' = 6C_1 x + 2C_2$$

$$-3D_1 x^2 l + D_2 x^2$$

$$w_d(l) = 0 \Rightarrow D_4 = 0$$

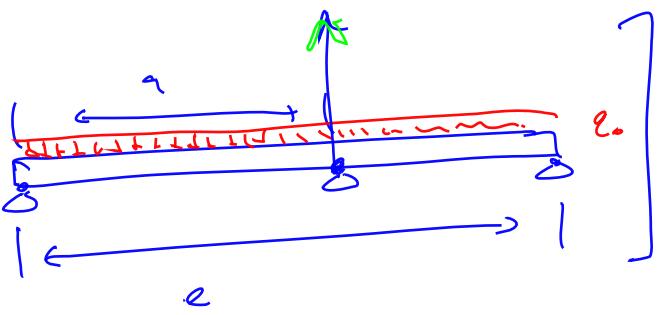
$$w_d' = 3D_1 (x-l)^2 + 2D_2 (x-l) + D_3$$

$$w_d'' = 6D_1 (x-l) + 2D_2 \Rightarrow D_2 = 0$$

$$a = \frac{l}{2}$$

Nosilci z večjim številom polji obremenitev.

Primer: Trotočkovno podprt nosilec.



$$0 \leq x \leq a \quad | \quad a \leq x \leq l$$

$$\omega_c \quad \omega_d$$

$$\omega_c = \frac{q_0}{24EI} x^4 + C_1 x^3 + C_2 x^2 + C_3 x + C_4$$

$$\omega_d = \frac{q_0}{24EI} x^4 + D_1 (x-a)^3 + D_2 (x-a)^2 + D_3 (x-a) + D_4$$

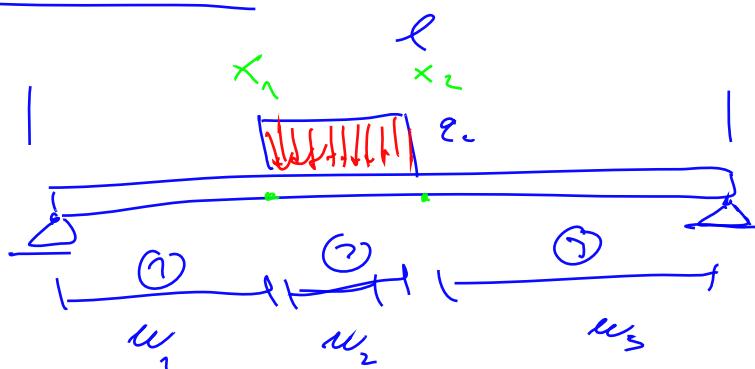
$$C_2 = C_4 = 0 ; \quad D_2 = D_4 = 0$$

$$\omega_d''' = \frac{q_0}{EI}$$

$$\omega_c(x=a) = 0 \quad \omega_d(x=a) = 0$$

$$\omega_c'(x=a) = \omega_d'(x=a)$$

$$\omega_c''(x=a) = \omega_d''(x=a)$$



$$\omega_1 = (C_1)x^3 + C_2 x^2 + (C_3)x + C_4 \quad ; \quad C_2 = C_4 = 0$$

$$\omega_3 = (D_1)(x-l)^3 + D_2 (x-l)^2 + (D_3)(x-l) + D_4 ; \quad D_2 = D_4 = 0$$

$$\omega_2 = \frac{q_0}{24EI} x^4 + (E_1)x^3 + (E_2)x^2 + (E_3)x + E_4$$

$$\omega_1(x_1) = \omega_2(x_1) ; \quad \omega_1'(x_1) = \omega_2'(x_1) ; \quad \omega_1''(x_1) = \omega_2''(x_1)$$

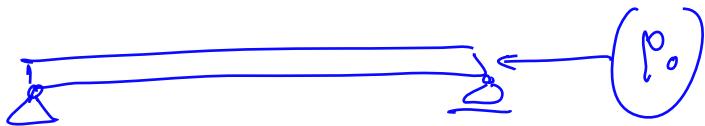
$$\omega_1'''(x_1) = \omega_2'''(x_1)$$

⁷ Podolžna = x₂

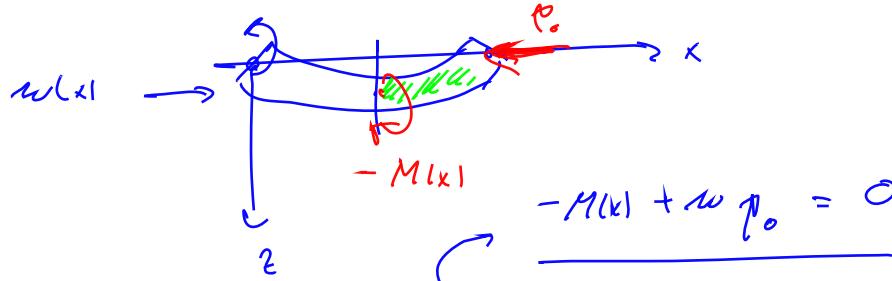
Metoda superpozicije.

Temperaturnii upogib nosilca, razlika temperature po prerezu, temperaturni moment.

Primer: konzolni nosilec, izračun upogiba za linerano razliko temperature po preseku.



Uklon nosilca.



$$w'' + \left(\frac{P_0}{EI}\right)w = 0$$

$$\begin{aligned} -M(x) + w P_0 &= 0 \\ M(x) &= -EIw'' \\ w(0) = 0; \quad w(l) &= 0 \end{aligned}$$

$$w = A \cos \sqrt{\frac{P_0}{EI}} x + B \sin \sqrt{\frac{P_0}{EI}} x \quad \checkmark$$

$$w' = -A \sqrt{\frac{P_0}{EI}} \sin \sqrt{\frac{P_0}{EI}} x + B \sqrt{\frac{P_0}{EI}} \cos \sqrt{\frac{P_0}{EI}} x$$

$$w'' = -A \left(\frac{P_0}{EI}\right) \cos \sqrt{\frac{P_0}{EI}} x - B \left(\frac{P_0}{EI}\right) \sin \sqrt{\frac{P_0}{EI}} x$$

$$w(x=c) = A \star \Rightarrow A = 0$$

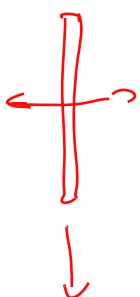
$$w(x=l) = B \sin \left(\sqrt{\frac{P_0}{EI}} l \right) = 0 \quad \underline{B = 0}$$

Izračun kritične obremenitve

$$P_c = \frac{\pi^2 EI}{l^2}$$

$$\sqrt{\frac{P_0}{EI}} l = \pi$$

$$P_c = \frac{\pi^2 EI}{l^2}$$



$$\frac{P_0}{EI} = \frac{\pi^2}{l^2}$$

$P_0 < P_c$ m. učink-

$P_0 \geq P_c$ most typi učink-

